

* Recap from last time

1) Generalized coordinates:

Not limited to formulating the problem in terms of orthogonal coordinate systems (cartesian, spherical, cyl., etc.)

$$\vec{r}_{\alpha,i} = \vec{r}_{\alpha,i}(q_1, \dots, q_{3N}, t) \quad \begin{array}{l} \alpha = 1, \dots, N \\ i = x, y, z \text{ or } r, \theta, \phi \text{ or } r, \phi, z, \sigma, \dots \end{array}$$

↓ invertible

$$q_j = q_j(\vec{r}_{\alpha,i}, t) \quad j = 1, \dots, S \quad (S = 3N)$$

$$\dot{q}_j = \dot{q}_j(\vec{r}_{\alpha,i}, \dot{\vec{r}}_{\alpha,i}, t)$$

* Still have the same Euler-Lagrange eqns. of motion

$$\frac{\partial L}{\partial q_j} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_j} = 0$$

generalized force: $Q_j \equiv \frac{\partial L}{\partial q_j}$

generalized momentum: $p_j \equiv \frac{\partial L}{\partial \dot{q}_j}$

ex: Consider 1 particle in 3d

$$Q_j = \frac{\partial L}{\partial q_j} = \sum_{\lambda=x,y,z} \frac{\partial L}{\partial r_\lambda} \frac{\partial r_\lambda}{\partial q_j} = \sum_{\lambda=x,y,z} F_\lambda \frac{\partial r_\lambda}{\partial q_j}$$

$$p_j = \frac{\partial L}{\partial \dot{q}_j} = \sum_{\lambda} \frac{\partial L}{\partial \dot{r}_\lambda} \frac{\partial \dot{r}_\lambda}{\partial \dot{q}_j} = \sum_{\lambda=x,y,z} p_\lambda \frac{\partial \dot{r}_\lambda}{\partial \dot{q}_j} = \sum_{\lambda} p_\lambda \frac{\partial r_\lambda}{\partial q_j}$$

used $\dot{r}_\lambda = \sum_{j=1}^S \frac{\partial r_\lambda}{\partial q_j} \dot{q}_j + \frac{\partial r_\lambda}{\partial t} \Rightarrow \frac{\partial \dot{r}_\lambda}{\partial \dot{q}_j} = \frac{\partial r_\lambda}{\partial q_j}$

* Problems w/ constraints

$$\text{eg. } J[q_1, q_2] = \int_{t_1}^{t_2} L(q_i, \dot{q}_i, t) dt$$

$$\text{w/ constraint } f(q_1, q_2, t) = 0$$

Issue: Can't blindly apply Hamilton's principle

$$\text{let } q_j(t, \alpha) = q_j(t) + \alpha \eta_j(t)$$

$$\eta_j(t_1) = \eta_j(t_2) = 0$$

$$\frac{\partial J}{\partial \alpha} = 0 = \int_{t_1}^{t_2} dt \left[\left(\frac{\partial L}{\partial q_1} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_1} \right) \eta_1(t) + \left(\frac{\partial L}{\partial q_2} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_2} \right) \eta_2(t) \right]$$

$$\text{* but } f(q_1, q_2) = 0 \Rightarrow df = 0 = \left(\frac{\partial f}{\partial q_1} \eta_1(t) + \frac{\partial f}{\partial q_2} \eta_2(t) \right) d\alpha$$

$$\Rightarrow \eta_2(t) = - \frac{\partial f / \partial q_1}{\partial f / \partial q_2} \eta_1(t) \quad \text{NOT INDEP! Can't set coeffs of } \eta_1, \eta_2 \text{ to zero!}$$

* 2 ways to deal w/ constraints in Lagrangian mechanics

1) Use constraint eqns. to eliminate "redundant" variables

$$\text{eg, for } s=2 \text{ case above, use } f(q_1, q_2, t) = 0 \Rightarrow \text{find } q_2 = q_2(q_1, t)$$

$$\Rightarrow L(q_1, q_2, \dot{q}_1, \dot{q}_2) \longrightarrow L(q_1, \dot{q}_1) \quad \text{expressed in the 1 independent coordinate}$$

$$\Rightarrow \text{Now apply Hamilton's Principle: } \frac{\partial L}{\partial q_1} - \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_1} \right) = 0$$

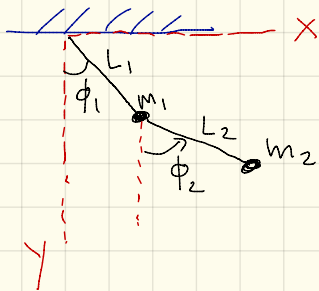
* Generalizing to $L(q_j, \dot{q}_j, t)$ $j=1, \dots, S$

w/ m eqns. of constraint $f_k(q_j, t) = 0$ $k=1, \dots, m$

\Rightarrow Use m constraint eqns to re-write L in terms of $(S-m)$ independent generalized coords

\Rightarrow then solve $\frac{\partial L}{\partial q_j} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_j} = 0$ $j=1, \dots, (S-m)$

Example: Co-planar double pendulum. Find L



2 masses \Rightarrow Naively, 6 DOF



Constrained motion

- planar x_1, x_2

- $L_1 + L_2$ fixed r_1, r_2

\therefore Really only 2 independent generalized coords. Needed.

\Rightarrow Try $\phi_1 + \phi_2$

$$T_1 = \frac{1}{2} m_1 L_1^2 \dot{\phi}_1^2$$

$$U_1 = -m_1 g L_1 \cos \phi_1$$

$$T_2 = \frac{m_2}{2} (\dot{x}_2^2 + \dot{y}_2^2)$$

$$U_2 = -m_2 g y_2$$

$$\text{but } x_2 = L_1 \sin \phi_1 + L_2 \sin \phi_2$$

$$y_2 = L_1 \cos \phi_1 + L_2 \cos \phi_2$$

$$\Rightarrow T_2 = \frac{1}{2} m_2 \left[L_1^2 \dot{\phi}_1^2 + L_2^2 \dot{\phi}_2^2 + 2L_1 L_2 \cos(\phi_1 - \phi_2) \dot{\phi}_1 \dot{\phi}_2 \right]$$

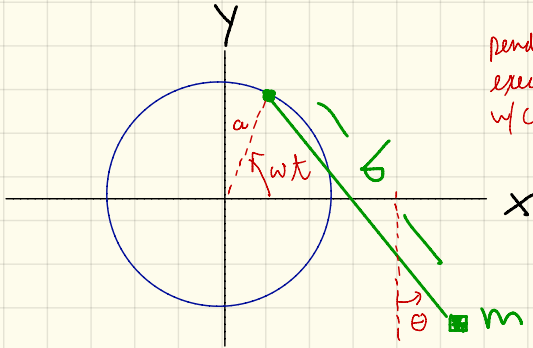
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$$L(\phi_1, \dot{\phi}_1, \phi_2, \dot{\phi}_2) = \frac{1}{2} (m_1 + m_2) L_1^2 \dot{\phi}_1^2 + \frac{1}{2} m_2 L_2^2 \dot{\phi}_2^2 + m_2 L_1 L_2 \dot{\phi}_1 \dot{\phi}_2 \cos(\phi_1 - \phi_2) \\ + (m_1 + m_2) g L_1 \cos \phi_1 + m_2 g L_2 \cos \phi_2$$

To find EOM, simply take $\frac{\partial L}{\partial \phi_i} - \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{\phi}_i} \right) = 0 \quad i=1,2$

Example

↓ g



pendulum support
executes circular motion
w/ constant ω + radius a

* Particle in 3D $\Rightarrow S=3$

* But planar motion + fixed $b \Rightarrow$ only $S-2=1$ independent coordinate Θ

$$x = a \cos \omega t + b \sin \Theta$$

$$y = a \sin \omega t - b \cos \Theta$$

$$\dot{x} = -a\omega \sin \omega t + b\dot{\Theta} \cos \Theta$$

$$\dot{y} = a\omega \cos \omega t + b\dot{\Theta} \sin \Theta$$

$$\Rightarrow T = \frac{m}{2} (\dot{x}^2 + \dot{y}^2) = \frac{m}{2} [a^2 \dot{\omega}^2 + b^2 \dot{\theta}^2 + 2b\dot{\theta} a \omega \sin(\theta - \omega t)]$$

$$U = mgy = mg(a \sin \omega t - b \cos \theta)$$

$$\Rightarrow L = T - U = \frac{m}{2} [a^2 \dot{\omega}^2 + b^2 \dot{\theta}^2 + 2b\dot{\theta} a \omega \sin(\theta - \omega t)] - mg(a \sin \omega t - b \cos \theta)$$

* 2nd Way to treat constraints: Lagrange Multipliers

* back to an generic constrained S=2 system

$$\frac{\partial J[q_1, q_2]}{\partial \alpha} = 0 = \int_{t_1}^{t_2} \left[\left(\frac{\partial L}{\partial q_1} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_1} \right) \eta_1(t) + \left(\frac{\partial L}{\partial q_2} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_2} \right) \eta_2(t) \right] dt$$

where $f(q_1, q_2, t) = 0 \Rightarrow \eta_2(t) = - \frac{\partial f / \partial q_1}{\partial f / \partial q_2} \eta_1(t)$

\therefore eliminating η_2 for $\eta_1 \Rightarrow$

$$\frac{\partial J}{\partial \alpha} = 0 = \int_{t_1}^{t_2} \left[\left(\frac{\partial L}{\partial q_1} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_1} \right) - \left(\frac{\partial L}{\partial q_2} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_2} \right) \frac{\partial f / \partial q_1}{\partial f / \partial q_2} \right] \eta_1(t) dt$$

$$\Rightarrow [] = 0 \Rightarrow \left(\frac{\partial L}{\partial q_1} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_1} \right) \left(\frac{\partial f}{\partial q_1} \right)^{-1} = \left(\frac{\partial L}{\partial q_2} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_2} \right) \left(\frac{\partial f}{\partial q_2} \right)^{-1}$$

$$\underbrace{\left(\frac{\partial L}{\partial q_1} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_1}\right)}_{\text{deriva. wrt } q_1, \dot{q}_1} \left(\frac{\partial f}{\partial q_1}\right)^{-1} = \underbrace{\left(\frac{\partial L}{\partial q_2} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_2}\right)}_{\text{deriva. wrt } q_2, \dot{q}_2} \left(\frac{\partial f}{\partial q_2}\right)^{-1}$$

$$L = L(q_1(t), \dot{q}_1(t), q_2(t), \dot{q}_2(t), t)$$

$$f = f(q_1(t), q_2(t), t)$$

\Rightarrow Only way this makes sense is if

$$\text{LHS} = -\lambda(t)$$

"Lagrange undetermined"

$$\text{RHS} = \text{LHS} = -\lambda(t)$$

"Multiplies"

\Downarrow

$$\frac{\partial L}{\partial q_1} - \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_1} \right) + \lambda(t) \frac{\partial f}{\partial q_1} = 0$$

$$\frac{\partial L}{\partial q_2} - \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_2} \right) + \lambda(t) \frac{\partial f}{\partial q_2} = 0$$

Plus the constraint eqn

$$f(q_1, q_2, t) = 0$$

\Rightarrow 3 eqns for 3 unknowns (q_1, q_2, λ)

* Claims (See the graduate text of Goldstein for a proof) that

$$Q_j^{\text{constraint}} = \lambda \frac{\partial f}{\partial q_j} = j^{\text{th}} \text{ component of the generalized force of constraint}$$

* Generalizing to higher # of DOF

$$L = L(q_j, \dot{q}_j; t) \quad j = 1, \dots, S$$

$$f_k(q_j; t) = 0 \quad k = 1, \dots, m$$

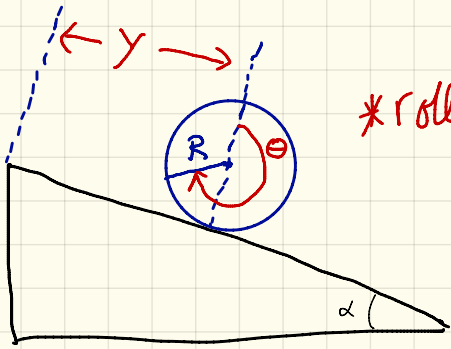
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$$\frac{\partial L}{\partial q_j} - \frac{d}{dt} \frac{\partial L}{\partial \dot{q}_j} + \sum_{k=1}^m \lambda_k(t) \frac{\partial f_k}{\partial q_j} = 0 \quad j = 1, \dots, S$$

$\Rightarrow (S+m)$ eqns for $(S+m)$ unknowns $(q_1, \dots, q_S; \lambda_1, \dots, \lambda_m)$

$$Q_j^{\text{const.}} = \sum_{k=1}^m \lambda_k \frac{\partial f_k}{\partial q_j} \quad \text{generalized forces of constraint}$$

Example: let's revisit the problem we solved with "Way 1" last time



* Rolling disk without slipping



Gen. Coords. y, θ

Constraint: $y = R\theta$

* Solve this w/ Lagrangian Multiplier + find Gen. Constraining Force

$$T = T_{\text{trans}} + T_{\text{rot}} = \frac{1}{2}M\dot{y}^2 + \frac{1}{2}I\dot{\theta}^2 \quad (I = \frac{1}{2}MR^2)$$

$$U = Mg(l-y)\sin\alpha \quad (l = \text{length of incline})$$

$$\textcircled{1} \quad \frac{\partial L}{\partial y} - \frac{d}{dt} \frac{\partial L}{\partial \dot{y}} + \lambda \frac{\partial f}{\partial y} = 0$$

and $f(y, \theta) = y - R\theta = 0$

$$\textcircled{2} \quad \frac{\partial L}{\partial \theta} - \frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} + \lambda \frac{\partial f}{\partial \theta} = 0$$

$$\Rightarrow \frac{\partial f}{\partial y} = 1, \quad \frac{\partial f}{\partial \theta} = -R$$

$$\begin{aligned} \textcircled{1} \Rightarrow & Mg\sin\alpha - M\ddot{y} + \lambda = 0 \\ \textcircled{2} \Rightarrow & -\frac{1}{2}MR^2\ddot{\theta} - \lambda R = 0 \end{aligned}$$

Lagrange plus $y = R\theta$
for y, θ, λ

$$y = R\theta \Rightarrow \ddot{\theta} = \frac{\ddot{y}}{R} \quad \text{plug into } \textcircled{2}$$

$$\textcircled{2} \Rightarrow -\frac{1}{2}MR\ddot{y} = \lambda R \Rightarrow \boxed{\lambda = -\frac{M}{2}\ddot{y}}$$

↓ plug into eq. ①

$$Mg\sin\alpha - M\ddot{y} - \frac{M}{2}\ddot{y} = 0$$

↓

$$\boxed{\ddot{y} = \frac{2g\sin\alpha}{3}}$$

Same ODE we got in the
Simpler way #1 of solving the problem

↓ plug into λ & $\ddot{\theta}$ eqns

$$\boxed{\lambda = -\frac{Mg\sin\alpha}{3}}$$

and

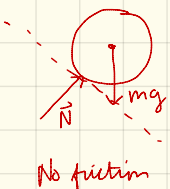
$$\boxed{\ddot{\theta} = \frac{2g\sin\alpha}{3R}}$$

* Forces of Constraint

$$Q_y = \lambda \frac{\partial F}{\partial y} = \lambda = -\frac{Mg}{3}\sin\alpha \quad (\text{Force})$$

$$Q_\theta = \lambda \frac{\partial F}{\partial \theta} = -\lambda R = \frac{MgR}{3}\sin\alpha \quad (\text{Torque})$$

Physical Interp: Frictional force is what implements the
"rolling w/out slipping" $y = R\theta$ constraint



disk will
slide (No rotation
since no torques)



torque
from
friction